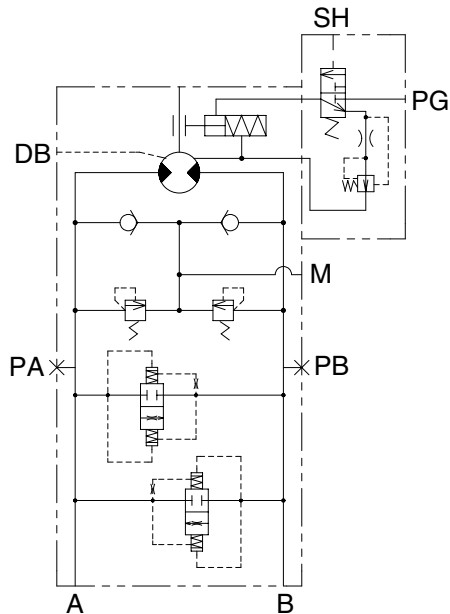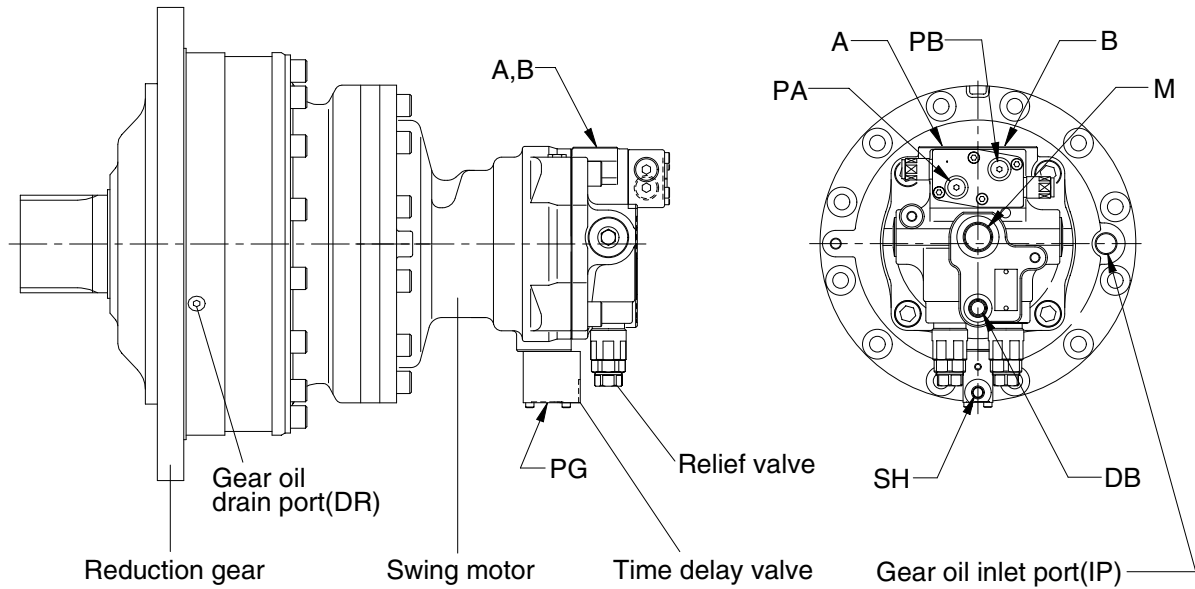# GROUP 3   SWING DEVICE(up to #0373)

1. STRUCTURE

Swing device consists swing motor, swing reduction gear.
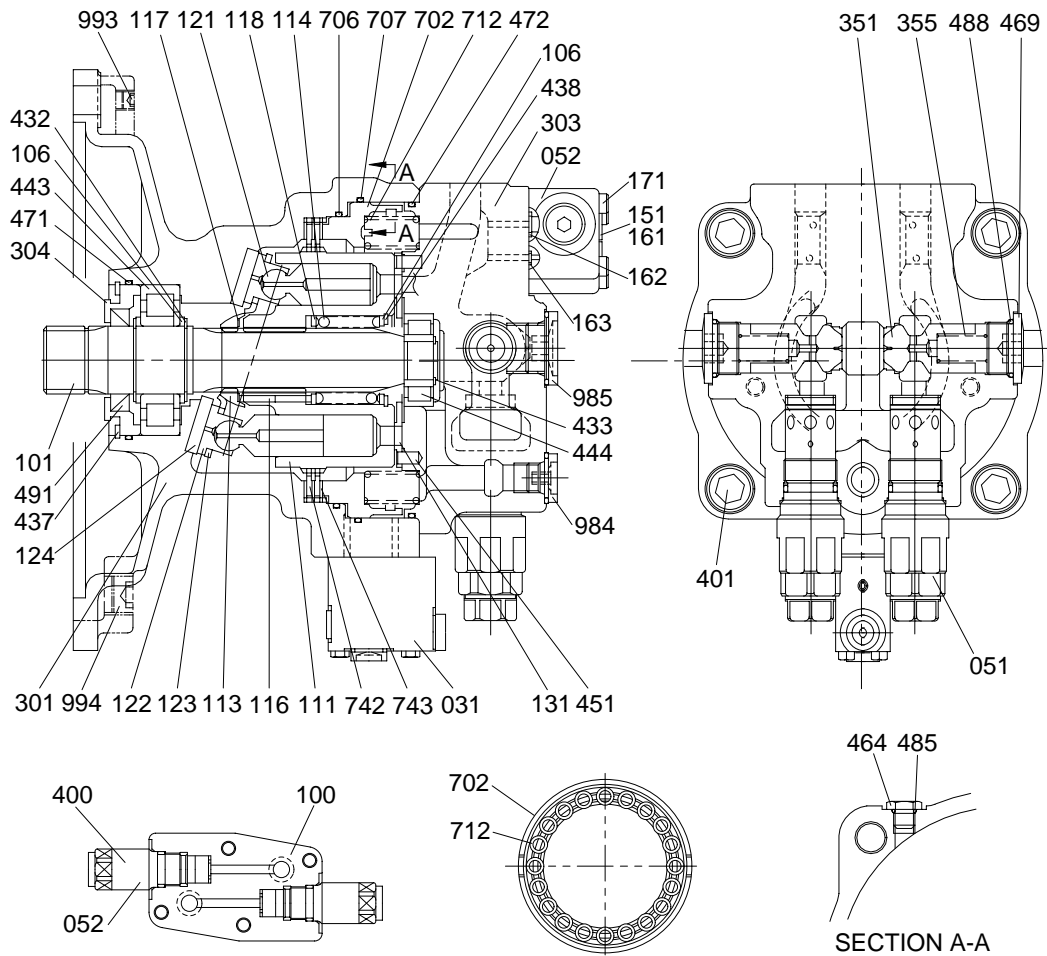
Swing motor include mechanical parking valve, relief valve, make up valve and time delay valve.

29072SM01

Port	Port name	Port size
A, B	Main port	⌀20
DB	Drain port	PF 1/2-19
M	Make up port	PF 1-24
PA, PB	Gauge port	PF 1/4-15
PG	Brake release port	PF 1/4-12
SH	Brake pilot port	PF 1/4-12
IP	Gear oil inlet port	PT 3/4-19
DR	Gear oil drain port	PT 1/2

1) SWING MOTOR

SECTION A-A

031	Time delay valve	161	O-ring	464	Plug
051	Relief valve	162	O-ring	469	Plug
052	Valve assy	163	O-ring	471	O-ring
100	Casing	171	Hexagon screw	472	O-ring
101	Drive shaft	301	Casing	485	O-ring
106	Spacer	303	Casing	488	O-ring
111	Cylinder	304	Front cover	491	Oil seal
113	Retainer	351	Plunger	702	Piston
114	Spring	355	Spring	706	O-ring
116	Push rod	400	Valve	707	O-ring
117	Spacer	401	Hexagon screw	712	Brake spring
118	Spacer	432	Snap ring	742	Lining plate
121	Piston	433	Snap ring	743	Separate plate
122	Shoe	437	Snap ring	984	Plug
123	Set plate	438	Snap ring	985	Plug
124	Shoe plate	443	Roller bearing	993	Plug
131	Valve plate	444	Needle bearing	994	Plug
151	Plug	451	Pin		

2) REDUCTION GEAR

1	Front casing	14	Pin 1	27	Oil seal
2	Middle casing	15	Side plate 1	28	Spring pin
3	Drive shaft	16	Magnet	29	Pressure plug
4	Ring gear 2	17	Side plate 3	30	Spring pin
5	Planet gear 2	18	Gauge bar	31	Stop ring
6	Sun gear 2	19	Gauge pipe	32	Bushing 2
7	Ring gear 1	20	Spacer ring	33	Stop ring
8	Planet gear 1	21	Knock pin	34	Lock washer
9	Sun gear 1	22	Roller bearing	35	Pinion gear
10	Carrier	23	Roller bearing	36	Lock plate
11	Pin 2	24	Needle cage	37	Hexagon bolt
12	Thrust washer	25	Socket bolt	38	Stop ring
13	Thrust button	26	Socket bolt	39	Side plate 2

2. FUNCTION

1) ROTARY PART

When high pressurized oil enters a cylinder through port(a), which is the inlet of balance plate(131), hydraulic pressure acting on the piston causes axial force F. The pressure force F works via the piston(121) upon the return plate(123) which acts upon the swash plate(124) via an hydrostatic bearing. Force F1 perpendicular to swash plate(124) and force F2 perpendicular to cylinder center. Being transferred to the cylinder block(111) through piston, force F2 causes rotational moment at surroundings of cylinder.

Since cylinder block has 9 equidistantly arrayed pistons, rotational torque is transmitted to cylinder shaft in order by several pistons connected to the inlet port of high pressurized oil. When the direction of oil flow is reversed, rotational direction of cylinder is also reversed. Output torque is given by the equation.

$$T = \frac{p \times q}{2}, \quad q = Z \cdot A \cdot PCD \cdot \tan, \quad F1 = \frac{F}{\cos}, \quad F_2 = F \tan, \quad S = PCD \times \tan$$

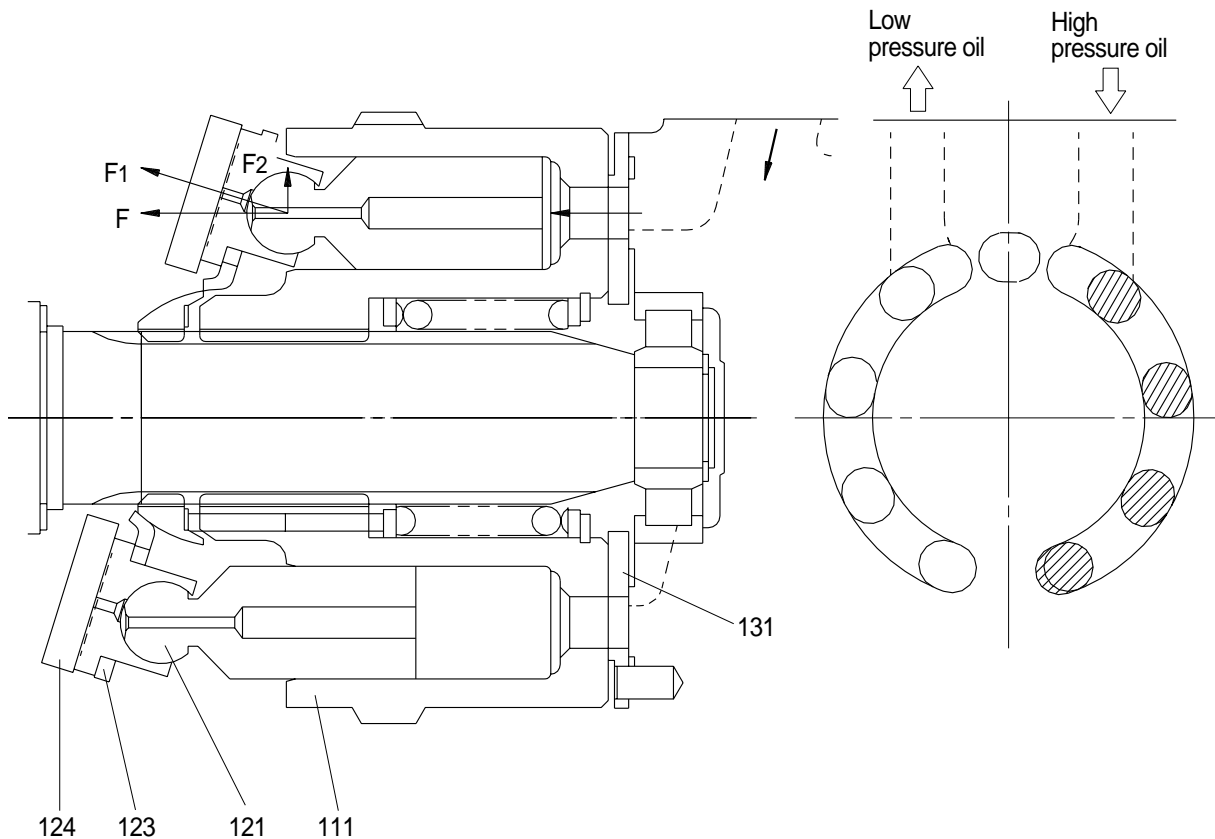Where p : Effective difference of pressure(kgf/cm²)

q : Displacement(cc/rev)

T : Output torque(kgf·cm)

Z : Piston number(9EA)

A : Piston area(cm²)

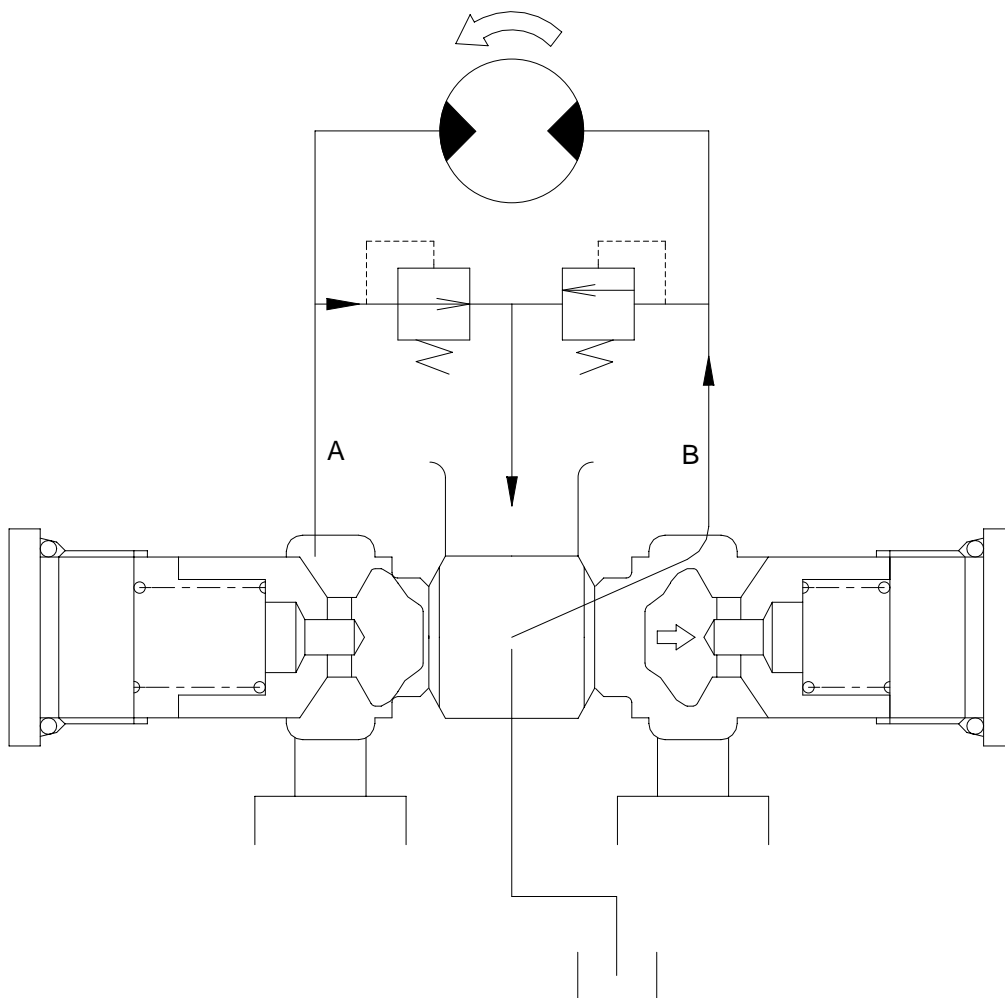: Tilting angle of swash plate(degree)

S : Piston stroke(cm)

2) MAKE UP VALVE

In the system using this type of motor, there is no counter balance functioning valve and there happens the case of revolution exceeding hydraulic supply of motor. To prevent the cavitation caused by insufficient oil flow there is a make up valve to fill up the oil insufficiency.
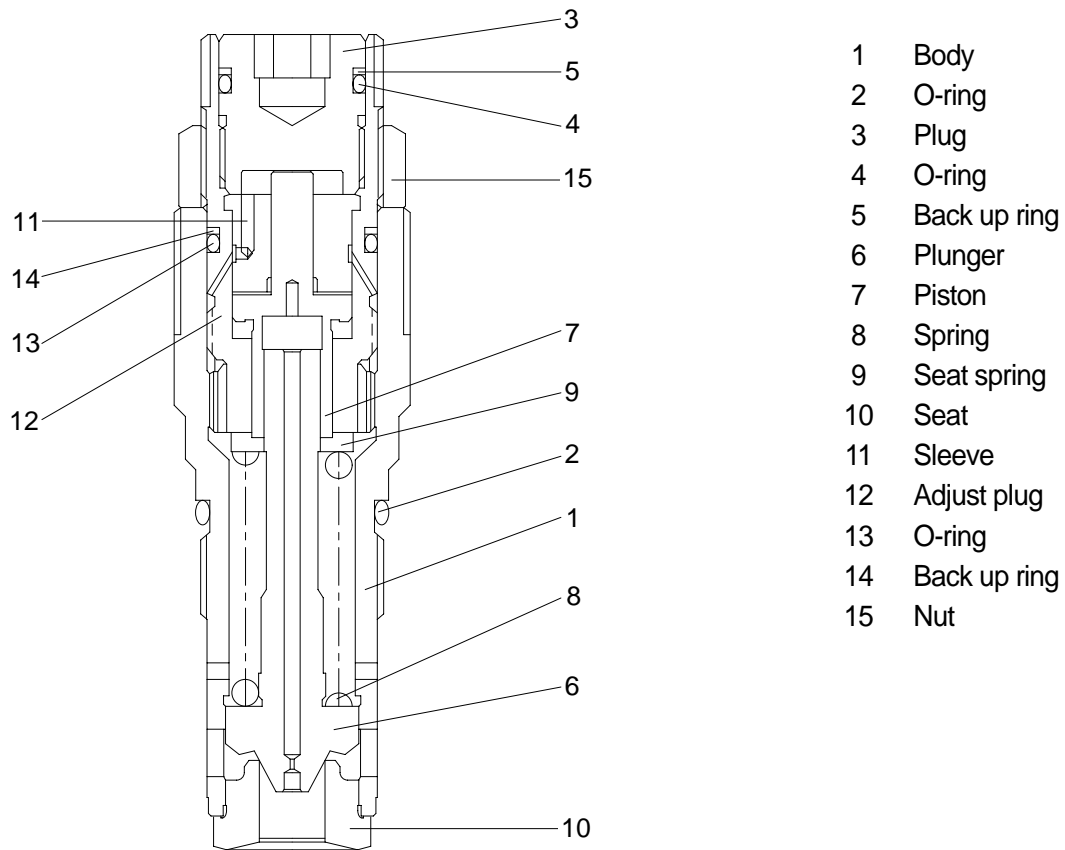
A make up valve is provided immediately before the port leading to the hydraulic oil tank to secure feed pressure required when the hydraulic motor makes a pumping action. The boost pressure acts on the hydraulic motor's feed port via the make up valve.

Pressurized oil into the port B, the motor rotate counterclockwise.

If the plunger of MCV moves neutral position, the oil in the motor is drain via left relief valve, the drain oil run into motor via right make up valve, which prevent the cavitation of motor.

3) RELIEF VALVE

|	
1	Body
2	O-ring
3	Plug
4	O-ring
5	Back up ring
6	Plunger
7	Piston
8	Spring
9	Seat spring
10	Seat
11	Sleeve
12	Adjust plug
13	O-ring
14	Back up ring
15	Nut

### (1) Construction of relief valve

The valve casing contains two cartridge type relief valves that stop the regular and reverse rotations of the hydraulic motor. The relief valves relieve high pressure at start or at stop of swing motion and can control the relief pressure in two steps, high and low, in order to insure smooth operation.

(2) Function of relief valve

Figure illustrates how the pressure acting on the relief valve is related to its rising process. Here is given the function, referring to the figure following page.
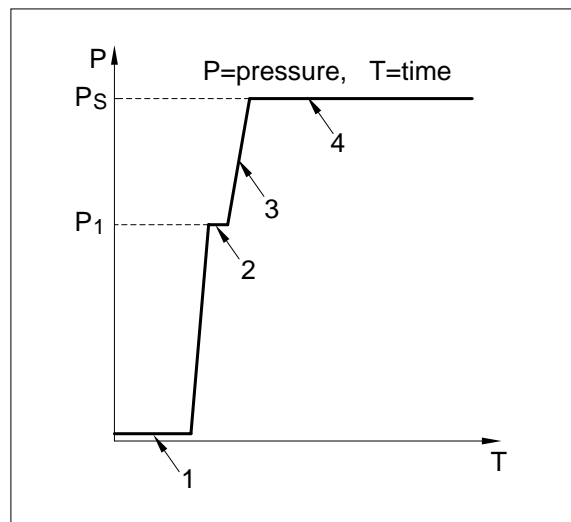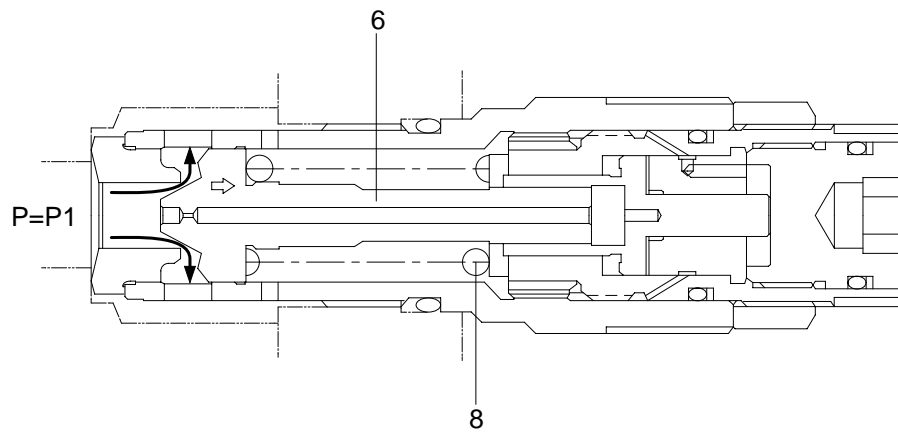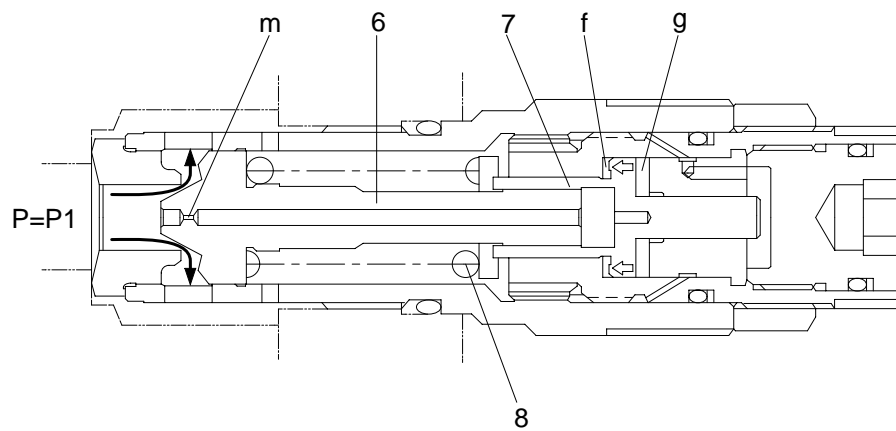
Ports (P,R) at tank pressure.
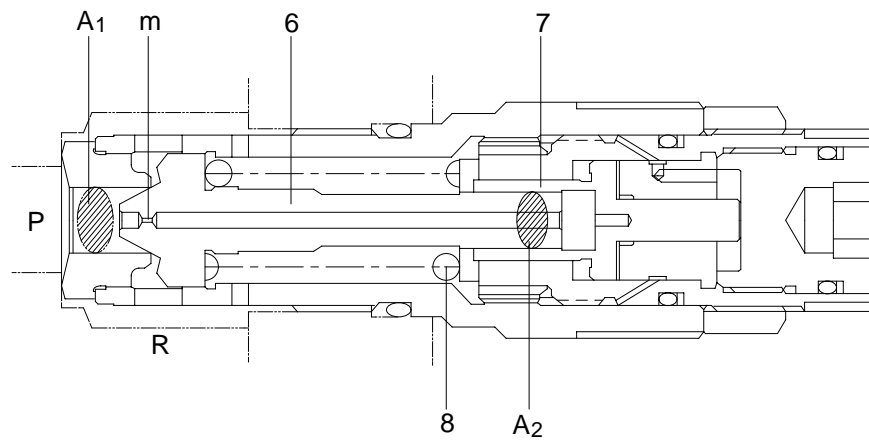
When hydraulic oil pressure($P \times A_1$) reaches the preset force(F_{SP}) of spring(8), the plunger (6) moves to the right as shown.

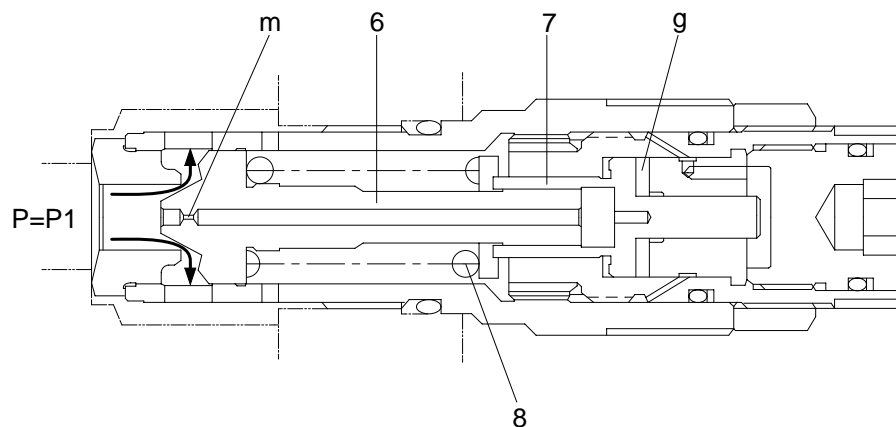$$P_1 \times A_1 = F_{sp} + P_1 \times A_2$$

$$P_1 = \frac{F_{sp}}{A_1 - A_2}$$

When the pressure of chamber g reaches the preset force(Fsp) of spring(8), the piston(7) moves right and stop the piston(7) hits the end of body.

When piston(7) hits the end of body, it stops moving to the right any further. As the result, the pressure in chamber(g) equals(Ps).

$$Ps \times A_1 = Fsp + Ps \times A_3$$

$$Ps = \frac{Fsp}{A_1 - A_3}$$

4) BRAKE SYSTEM

(1) Control valve swing brake system
This is the brake system to stop the swing motion of the excavator during operation.

In this system, the hydraulic circuit is throttled by the swing control valve, and the resistance created by this throttling works as a brake force to slow down the swing motion.
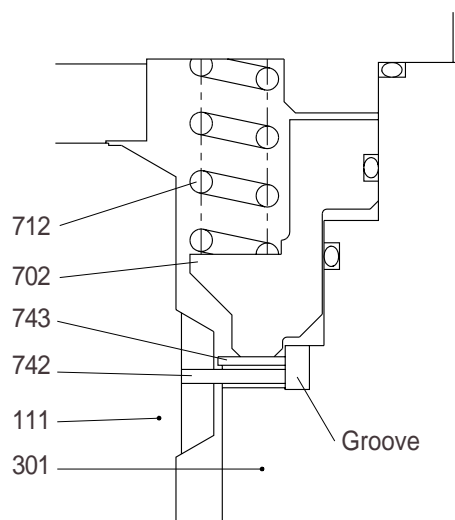

Work	Deceleration	Stop
MCV A, B opened	MCV A, B throttled	MCV A, B closed
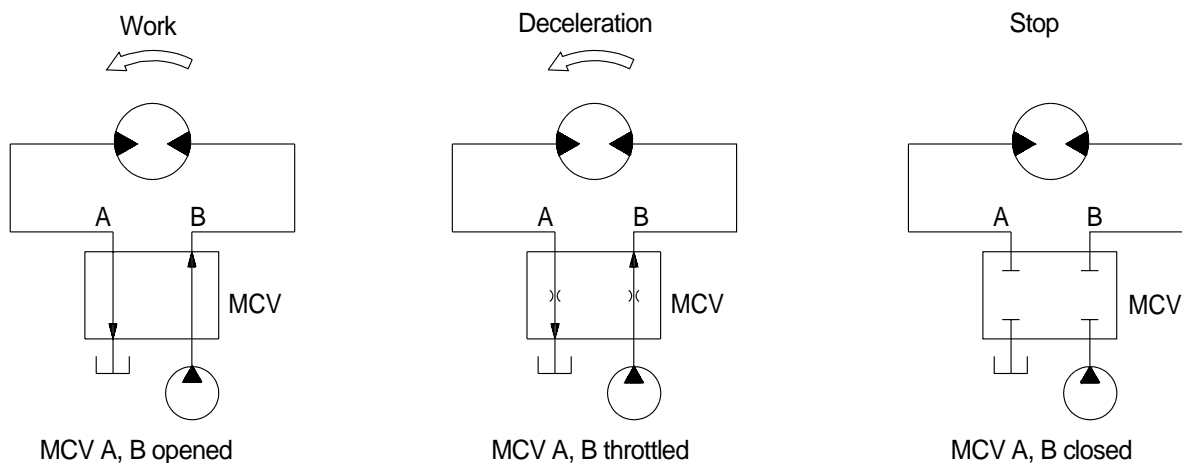
(2) Mechanical swing parking brake system
The mechanical swing parking brake system is installed to prevent the upper structure from swinging downhill because of its own weight when the excavator is parked on a slope since it completely eliminates the hydraulic drift of swing motion while the excavator is on a slop, work can be done more easily and safely.

Brake assembly

Circumferential rotation of separate plate(743) is constrained by the groove located at casing(301). When housing is pressed down by brake spring(712) through lining plate(742), separate plate(743) and brake piston(702), friction force occurs there.

Cylinder(111) is constrained by this friction force and brake acts, while brake releases when hydraulic force exceeds spring force.
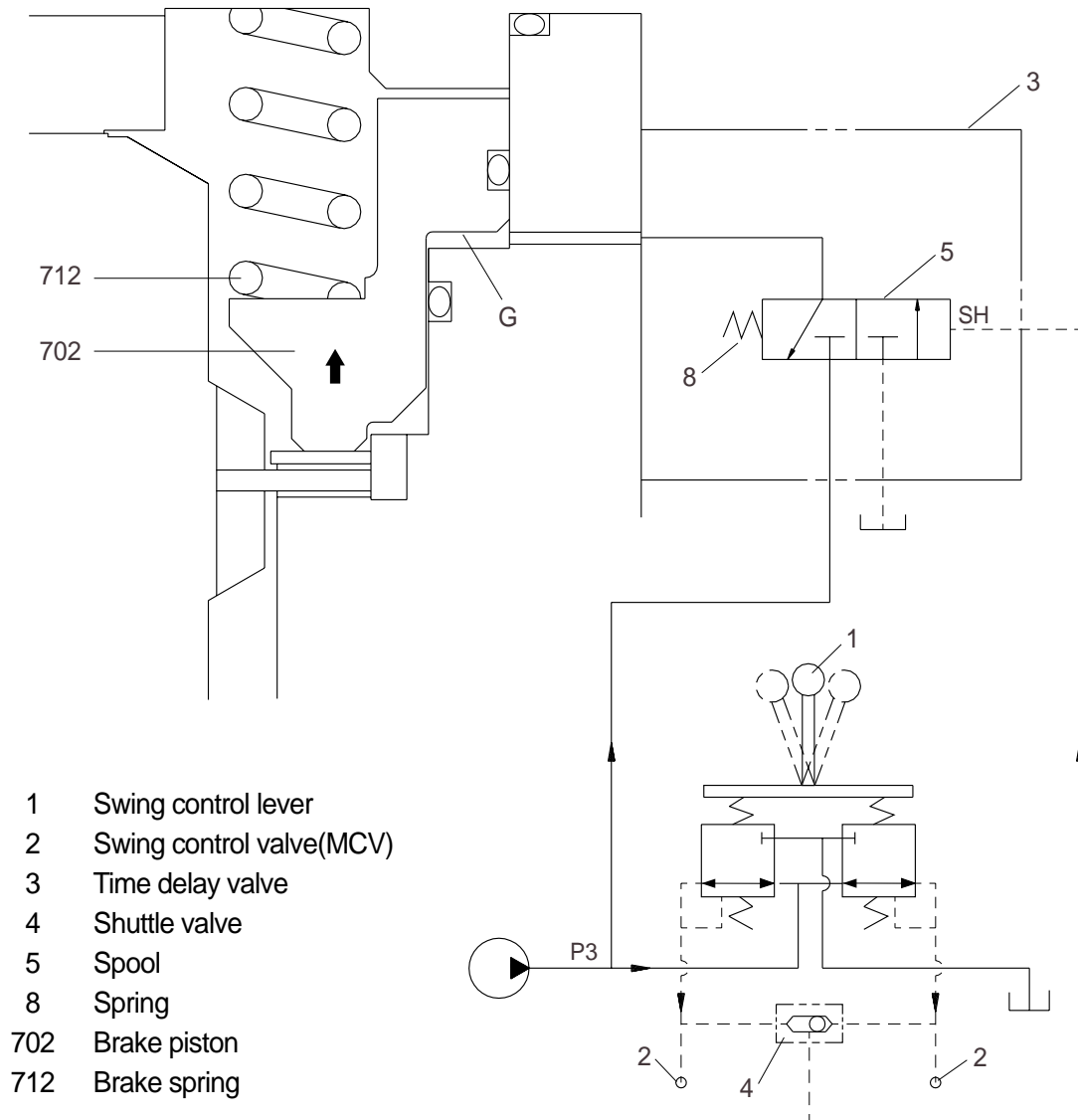

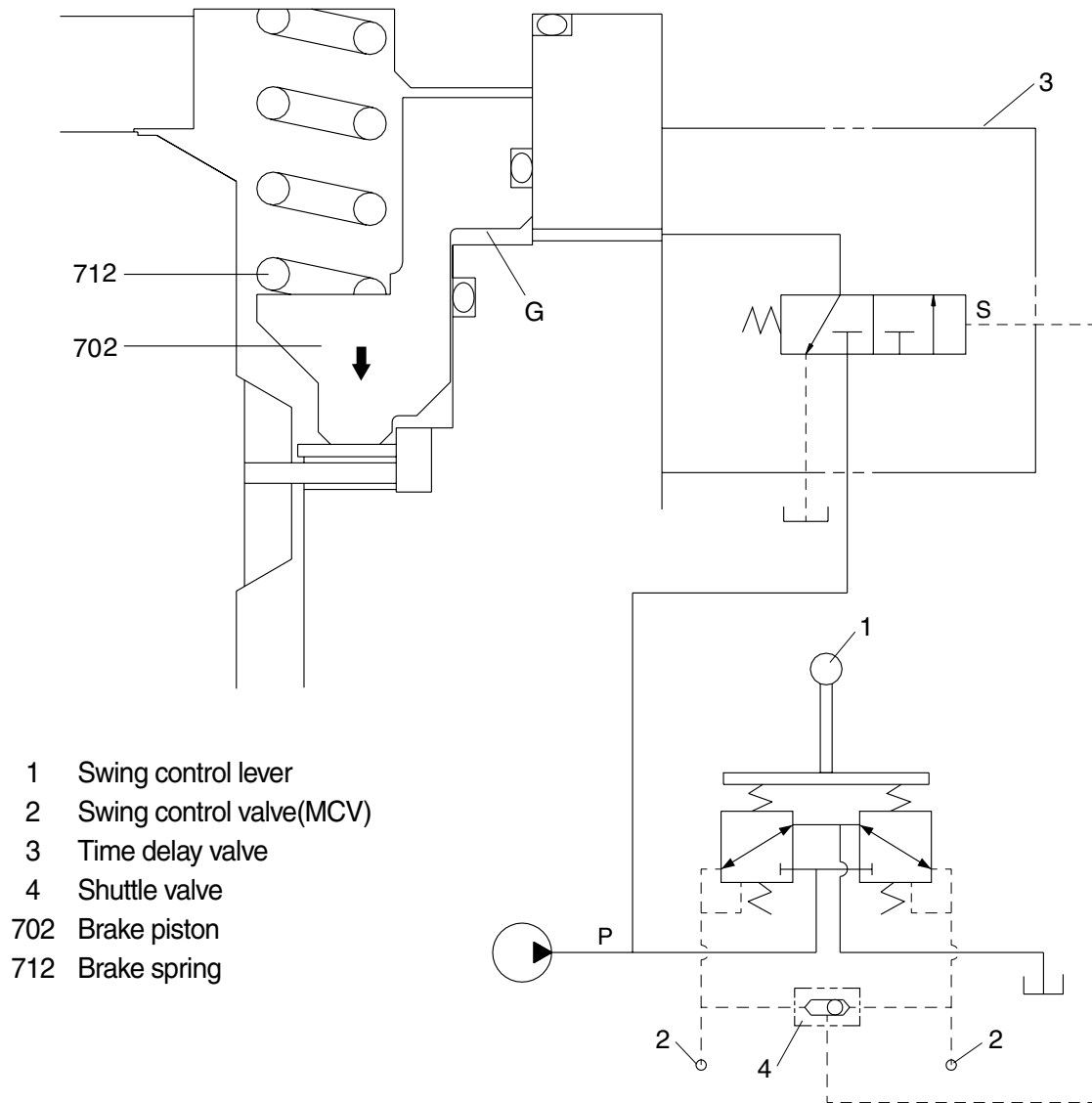| 111	Cylinder	712	Brake spring
301	Casing	742	Lining plate
702	Brake piston	743	Separate plate

Operating principle

a. When the swing control lever(1) is set to the swing position, the pilot oil go to the swing control valve(2) and to SH of the time delay valve(3) via the shuttle valve(4), this pressure move spool(5) to the leftward against the force of the spring(8), so pilot pump charged oil(P3) goes to the chamber G.

This pressure is applied to move the piston(702) to the upward against the force of the spring(712). Thus, it releases the brake force.

1	Swing control lever
2	Swing control valve(MCV)
3	Time delay valve
4	Shuttle valve
5	Spool
8	Spring
702	Brake piston
712	Brake spring

b. When the swing control lever(1) is set the neutral position, the time delay valve(3) shifts the neutral position and the pilot oil blocked chamber G.
Then, the piston(702) is moved lower by spring(712) force and the return oil from the chamber G is drain.

1   Swing control lever
2   Swing control valve(MCV)
3   Time delay valve
4   Shuttle valve
702 Brake piston
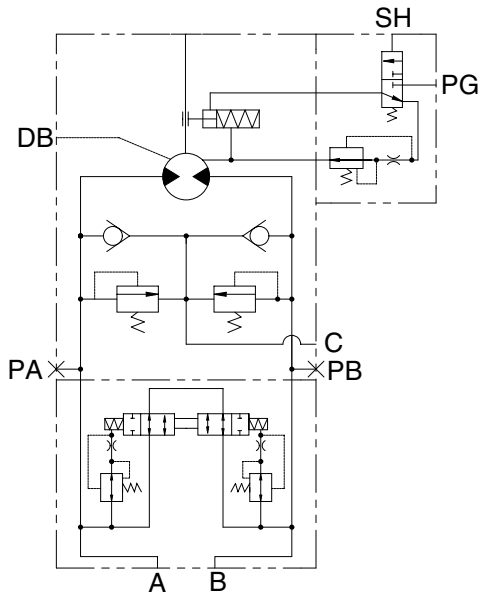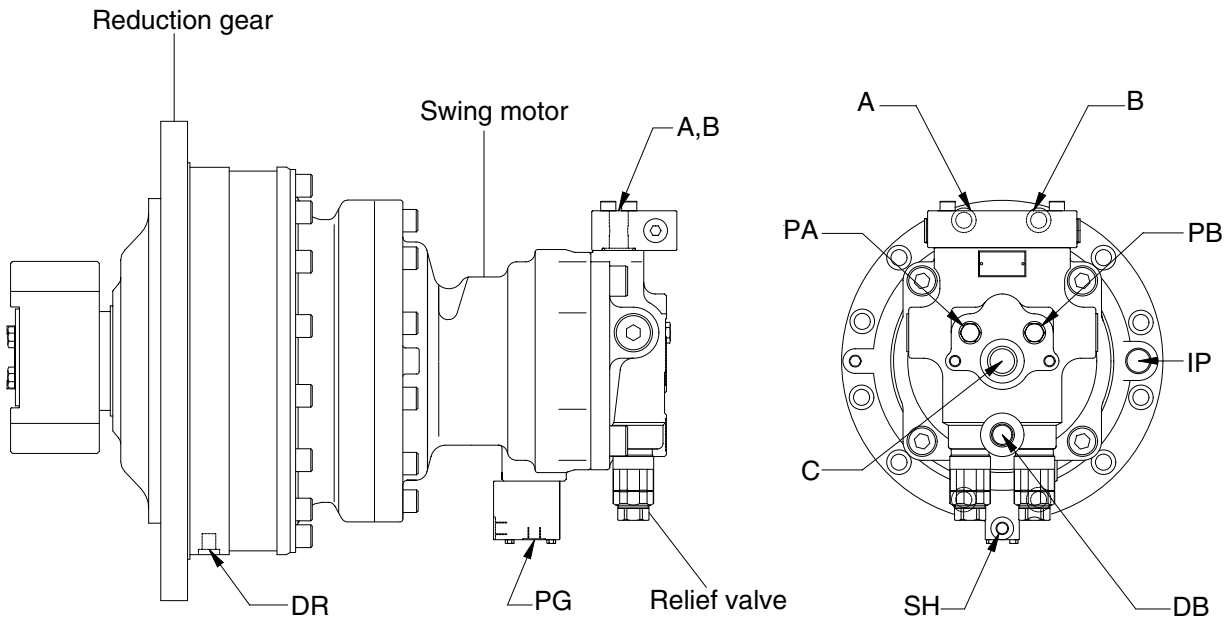712 Brake spring

R290SM08(1)

GROUP 3   SWING DEVICE(#0374 and up)

1. STRUCTURE

Swing device consists swing motor, swing reduction gear.

Swing motor include mechanical parking valve, relief valve, make up valve and time delay valve.

Port	Port name	Port size
A, B	Main port	∅ 20
DB	Drain port	PF 1/2-19
C	Make up port	PF 1-24
PA, PB	Gauge port	PF 1/4-15
PG	Brake release port	PF 1/4-12
SH	Brake pilot port	PF 1/4-12
IP	Gear oil inlet port	PT 3/4-19
DR	Gear oil drain port	PT 1/2

30572SM01

1) SWING MOTOR
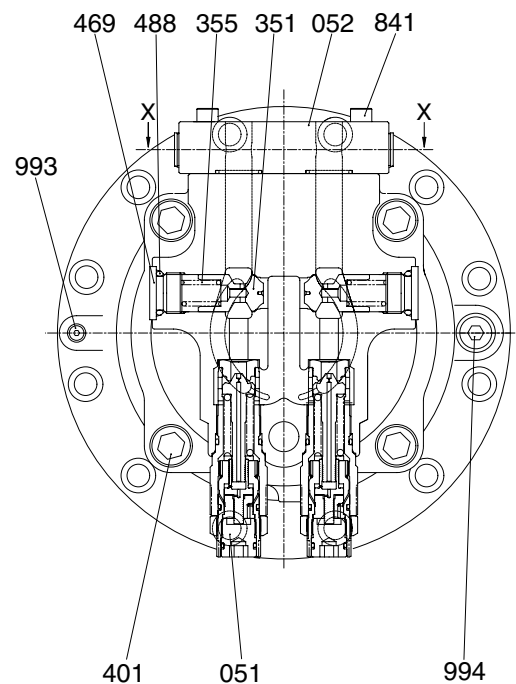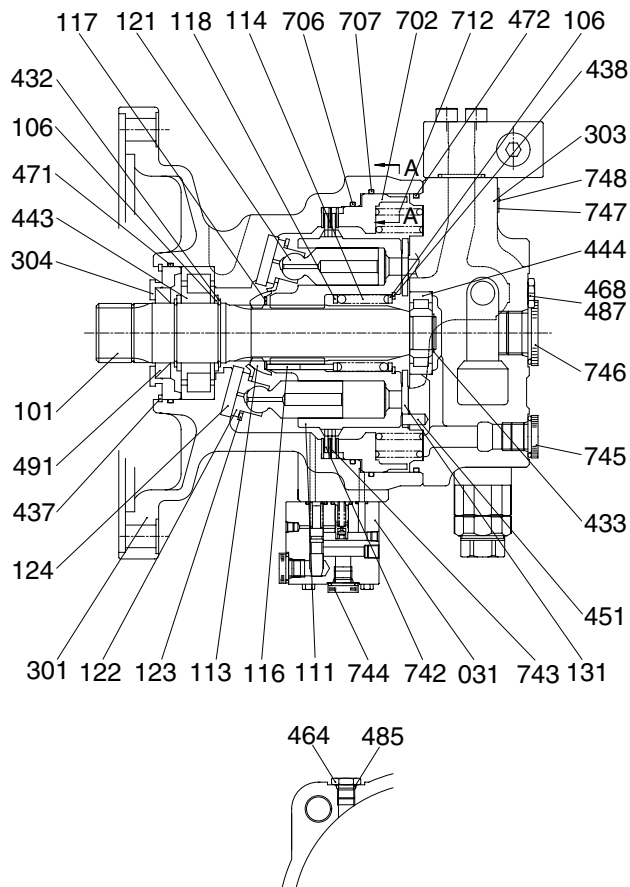
30572SM02

031	Brake valve	304	Front cover	488	O-ring
051	Relief valve	351	Plunger	491	Oil seal
052	Valve assy	355	Spring	702	Piston
101	Drive shaft	401	Socket bolt	706	O-ring
106	Spacer	432	Snap ring	707	O-ring
111	Cylinder block	433	Snap ring	712	Brake spring
113	Bushing	437	Snap ring	742	Lining plate
114	Spring	438	Snap ring	743	Separate plate
116	Push rod	443	Roller bearing	744	Plug
117	Spacer	444	Roller bearing	745	Plug
118	Spacer	451	Spring pin	746	Plug
121	Piston	464	Plug	747	Name plate
122	Shoe	468	Plug	748	Rivet screw
123	Retainer	469	Plug	841	Socket bolt
124	Shoe plate	471	O-ring	993	Plug
131	Valve plate	472	O-ring	994	Plug
301	Casing	485	O-ring		
303	Casing	487	O-ring		

2) REDUCTION GEAR
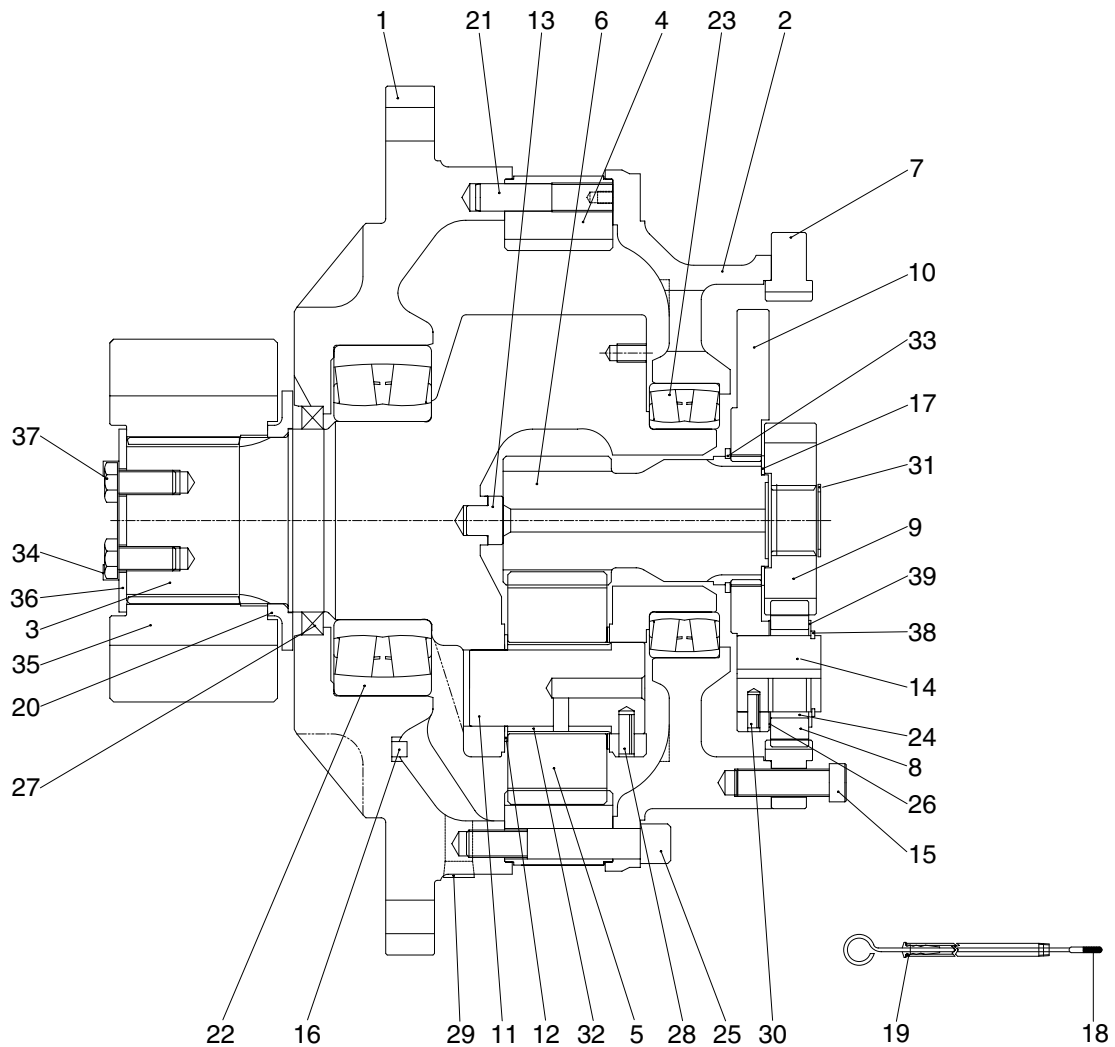
30572SR01

1	Front casing	14	Pin 1	27	Oil seal
2	Middle casing	15	Side plate 1	28	Spring pin
3	Drive shaft	16	Magnet	29	Pressure plug
4	Ring gear 2	17	Side plate 3	30	Spring pin
5	Planet gear 2	18	Gauge bar	31	Stop ring
6	Sun gear 2	19	Gauge pipe	32	Bushing 2
7	Ring gear 1	20	Spacer ring	33	Stop ring
8	Planet gear 1	21	Knock pin	34	Lock washer
9	Sun gear 1	22	Roller bearing	35	Pinion gear
10	Carrier	23	Roller bearing	36	Lock plate
11	Pin 2	24	Needle cage	37	Hexagon bolt
12	Thrust washer	25	Socket bolt	38	Stop ring
13	Thrust button	26	Socket bolt	39	Side plate 2

2. FUNCTION

1) ROTARY PART

When high pressurized oil enters a cylinder through port(a), which is the inlet of balance plate(131), hydraulic pressure acting on the piston causes axial force F. The pressure force F works via the piston(121) upon the return plate(123) which acts upon the swash plate(124) via an hydrostatic bearing. Force F1 perpendicular to swash plate(124) and force F2 perpendicular to cylinder center.

Being transferred to the cylinder block(111) through piston, force F2 causes rotational moment at surroundings of cylinder.

Since cylinder block has 9 equidistantly arrayed pistons, rotational torque is transmitted to cylinder shaft in order by several pistons connected to the inlet port of high pressurized oil. When the direction of oil flow is reversed, rotational direction of cylinder is also reversed. Output torque is given by the equation.

$$T = \frac{p \times q}{2\pi}, \quad q = Z \cdot A \cdot PCD \cdot \tan\theta, \quad F1 = \frac{F}{\cos\theta}, \quad F_2 = F\tan\theta, \quad S = PCD \times \tan\theta$$

Where p : Effective difference of pressure(kgf/cm²)
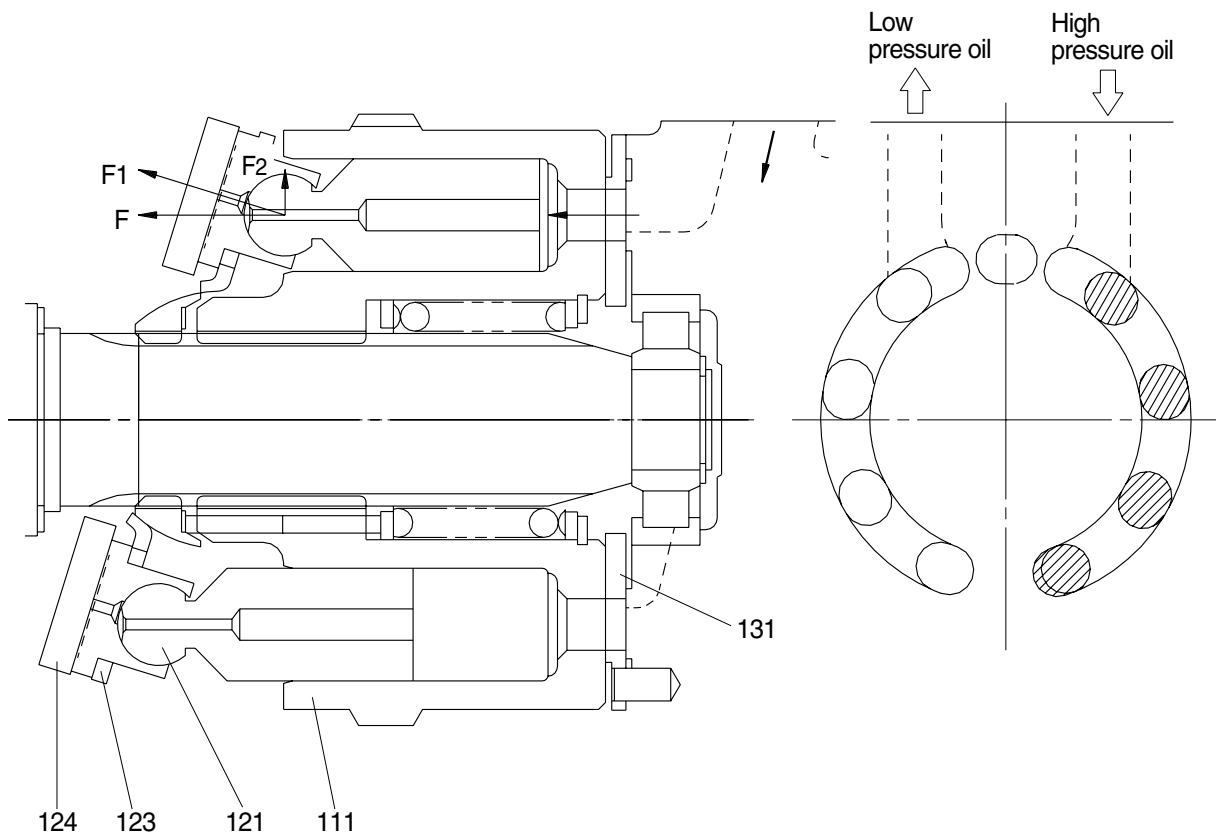
      q : Displacement(cc/rev)

      T : Output torque(kgf · cm)

      Z : Piston number(9EA)

      A : Piston area(cm²)

      θ : Tilting angle of swash plate(degree)
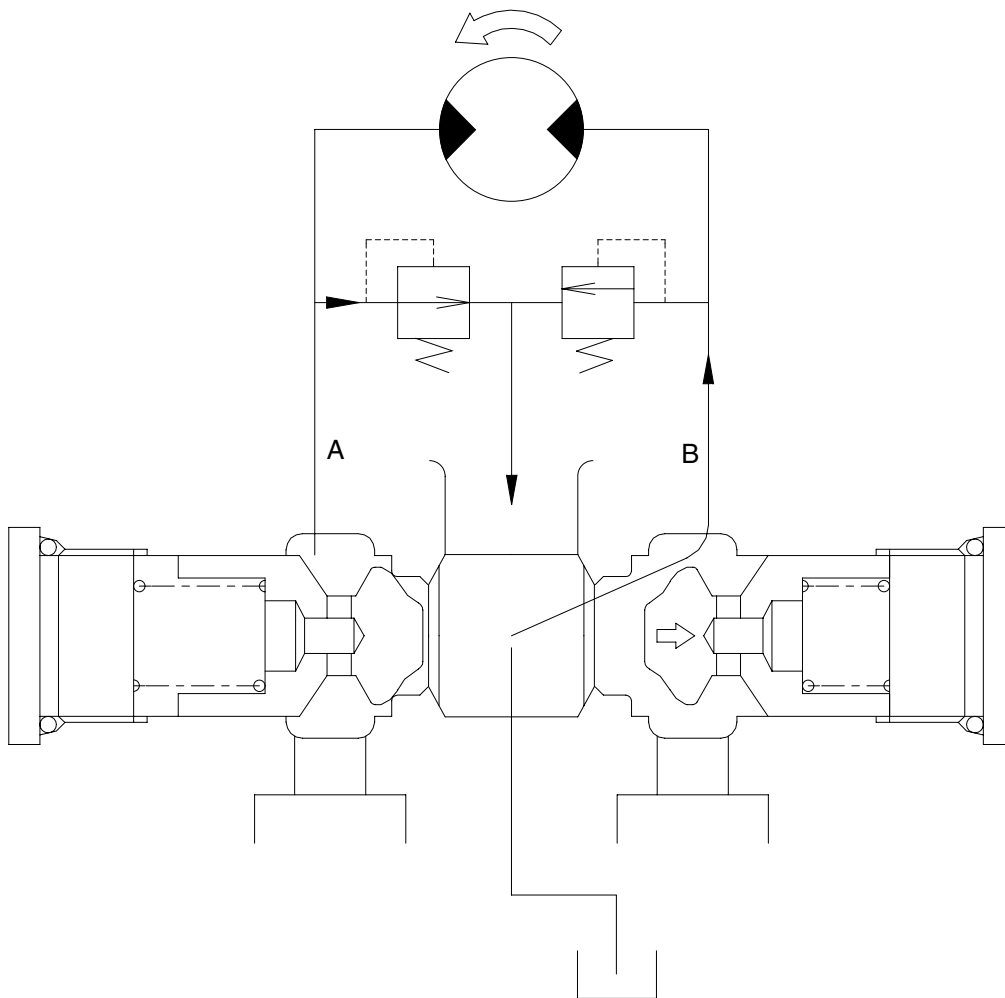
      S : Piston stroke(cm)

R290SM05(2)

2) MAKE UP VALVE

In the system using this type of motor, there is no counter balance functioning valve and there happens the case of revolution exceeding hydraulic supply of motor. To prevent the cavitation caused by insufficient oil flow there is a make up valve to fill up the oil insufficiency.

A make up valve is provided immediately before the port leading to the hydraulic oil tank to secure feed pressure required when the hydraulic motor makes a pumping action. The boost pressure acts on the hydraulic motor's feed port via the make up valve.

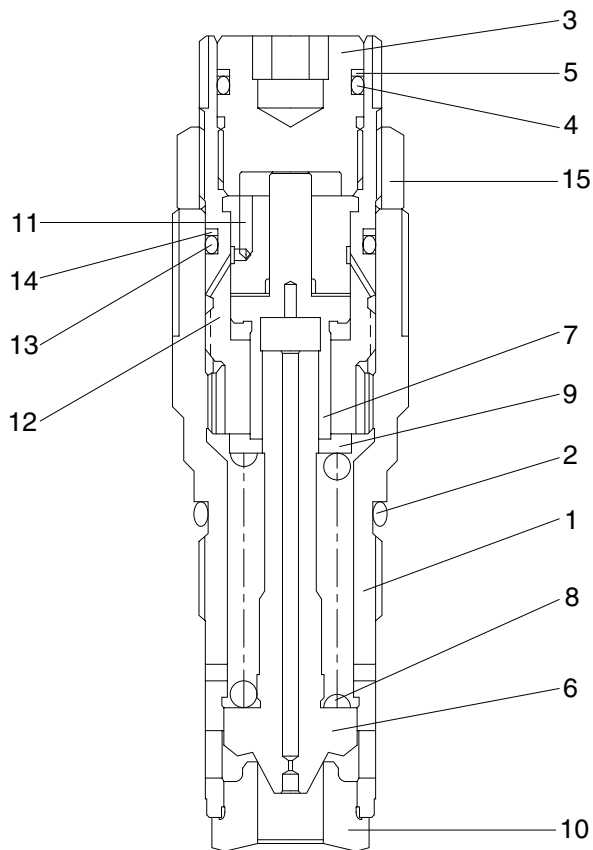Pressurized oil into the port B, the motor rotate counterclockwise.

If the plunger of MCV moves neutral position, the oil in the motor is drain via left relief valve, the drain oil run into motor via right make up valve, which prevent the cavitation of motor.
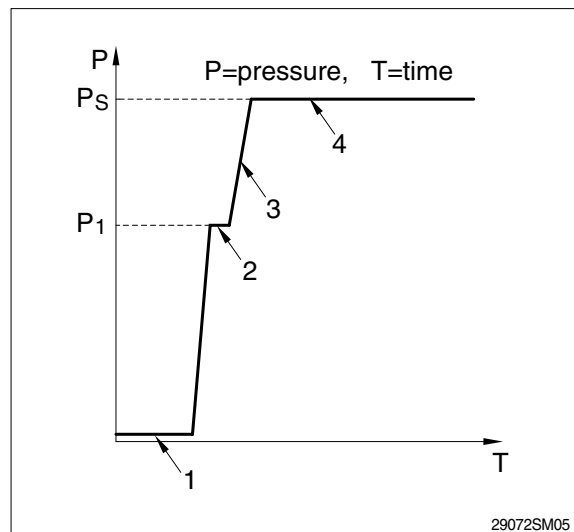

29072SM09

3) RELIEF VALVE

1	Body
2	O-ring
3	Plug
4	O-ring
5	Back up ring
6	Plunger
7	Piston
8	Spring
9	Seat spring
10	Seat
11	Sleeve
12	Adjust plug
13	O-ring
14	Back up ring
15	Nut

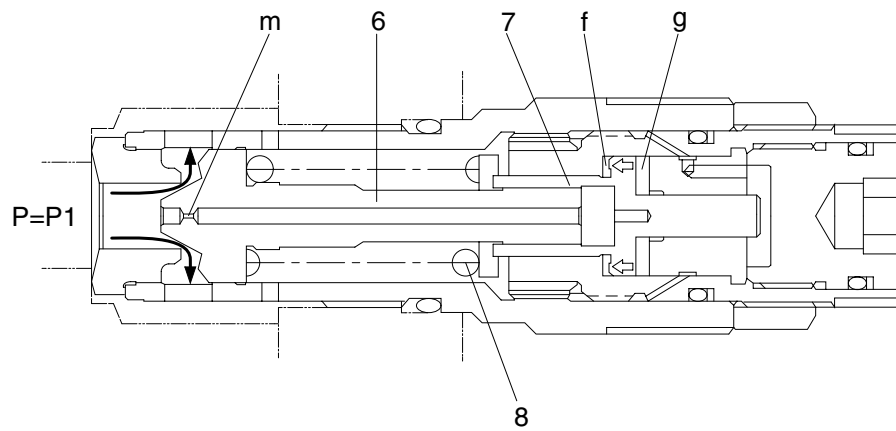### (1) Construction of relief valve

The valve casing contains two cartridge type relief valves that stop the regular and reverse rotations of the hydraulic motor. The relief valves relieve high pressure at start or at stop of swing motion and can control the relief pressure in two steps, high and low, in order to insure smooth operation.

(2) Function of relief valve

Figure illustrates how the pressure acting on the relief valve is related to its rising process. Here is given the function, referring to the figure following page.

29072SM05

③ When the pressure of chamber g reaches the preset force(Fsp) of spring(8), the piston(7) moves right and stop the piston(7) hits the end of body.
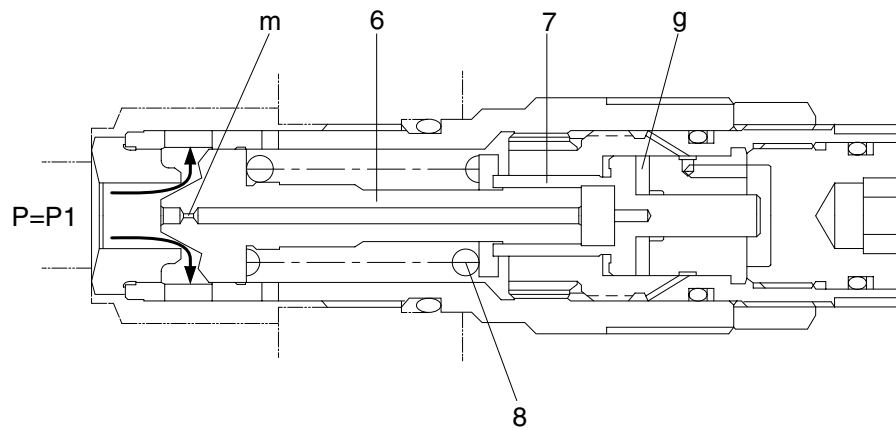

P=P1

29072SM07

④ When piston(7) hits the end of body, it stops moving to the right any further.   As the result, the pressure in chamber(g) equals(Ps).

$$Ps \times A_1 = Fsp + Ps \times A_3$$

$$Ps = \frac{Fsp}{A_1 - A_3}$$

P=P1

29072SM08

4) BRAKE SYSTEM

(1) Control valve swing brake system

This is the brake system to stop the swing motion of the excavator during operation.

In this system, the hydraulic circuit is throttled by the swing control valve, and the resistance created by this throttling works as a brake force to slow down the swing motion.

Work	Deceleration	Stop
A B	A B	A B
MCV	MCV	MCV
MCV A, B opened	MCV A, B throttled	MCV A, B closed
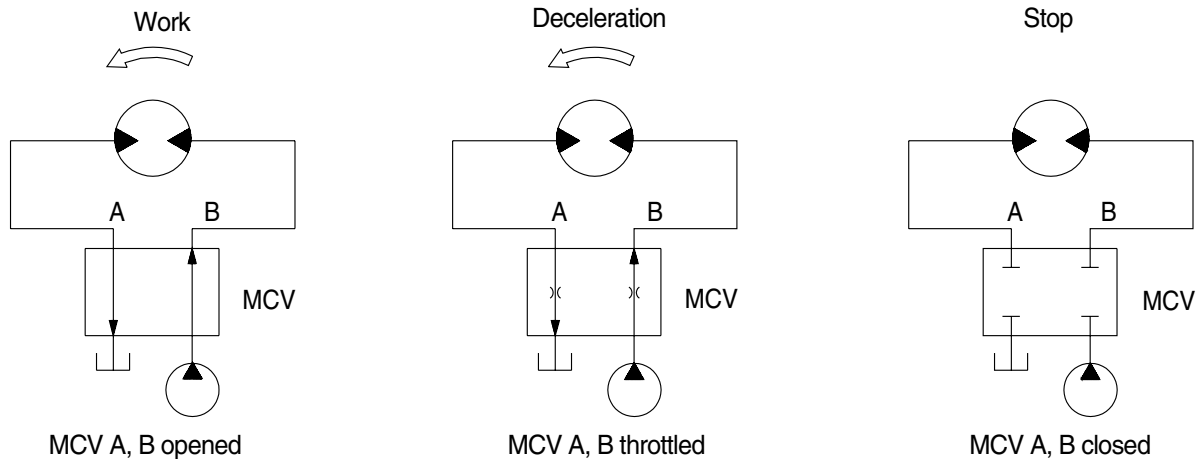
(2) Mechanical swing parking brake system

The mechanical swing parking brake system is installed to prevent the upper structure from swinging downhill because of its own weight when the excavator is parked on a slope since it completely eliminates the hydraulic drift of swing motion while the excavator is on a slop, work can be done more easily and safely.

① Brake assembly

Circumferential rotation of separate plate(743) is constrained by the groove located at casing(301). When housing is pressed down by brake spring(712) through lining plate(742), separate plate(743) and brake piston(702), friction force occurs there.

Cylinder(111) is constrained by this friction force and brake acts, while brake releases when hydraulic force exceeds spring force.

712
702
743
742
111
301
Groove
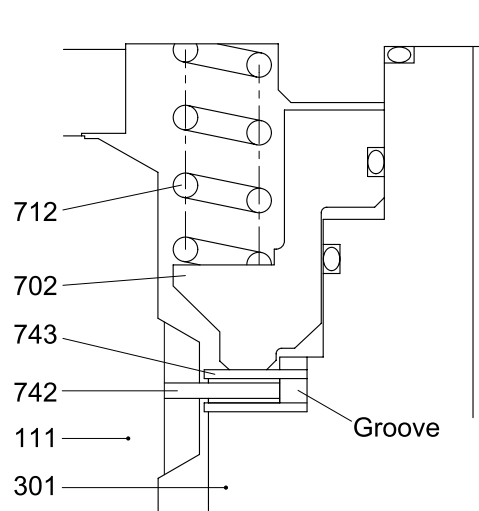
30572SM03

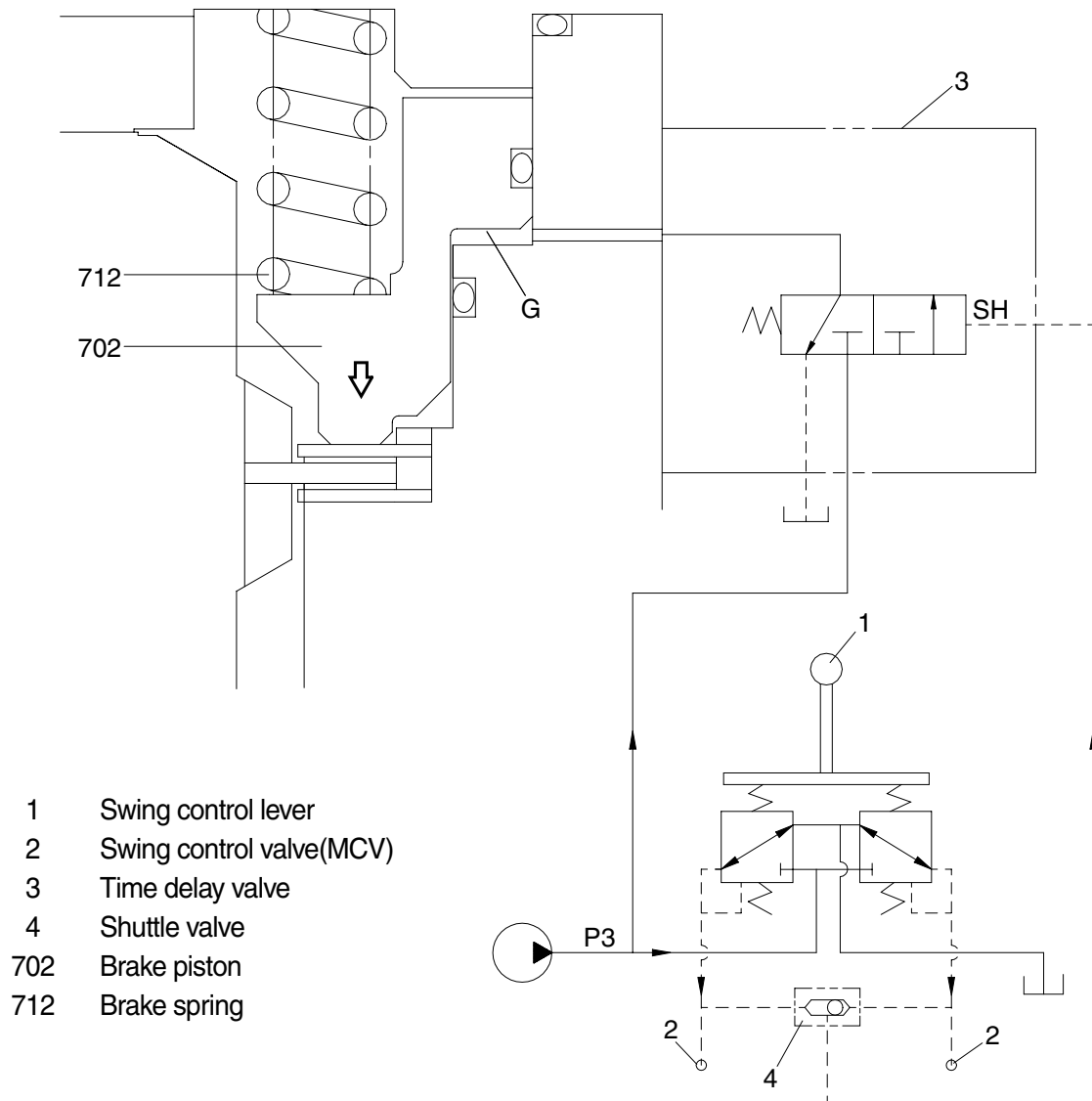| 111	Cylinder	712	Brake spring
301	Casing	742	Lining plate
702	Brake piston	743	Separate plate

② Operating principle

a. When the swing control lever(1) is set to the swing position, the pilot oil go to the swing control valve(2) and to SH of the time delay valve(3) via the shuttle valve(4), this pressure move spool(5) to the leftward against the force of the spring(8), so pilot pump charged oil(P3) goes to the chamber G.

This pressure is applied to move the piston(702) to the upward against the force of the spring(712). Thus, it releases the brake force.

1	Swing control lever
2	Swing control valve(MCV)
3	Time delay valve
4	Shuttle valve
5	Spool
8	Spring
702	Brake piston
712	Brake spring
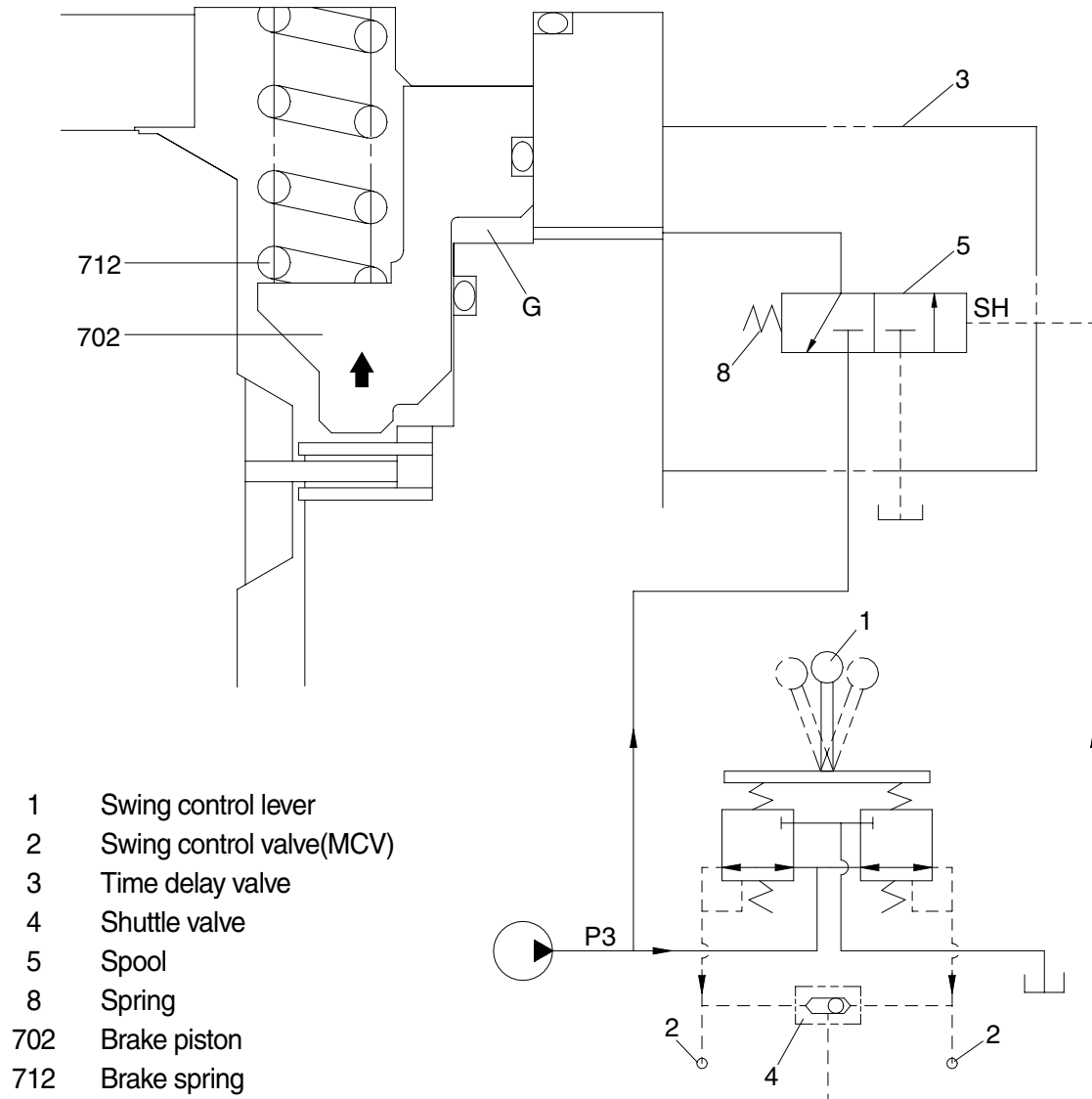
30572SM04

b. When the swing control lever(1) is set the neutral position, the time delay valve(3) shifts the neutral position and the pilot oil blocked chamber G.

Then, the piston(702) is moved lower by spring(712) force and the return oil from the chamber G is drain.

1	Swing control lever
2	Swing control valve(MCV)
3	Time delay valve
4	Shuttle valve
702	Brake piston
712	Brake spring

30572SM05